

**CDF
CMEX
UPGRADE**

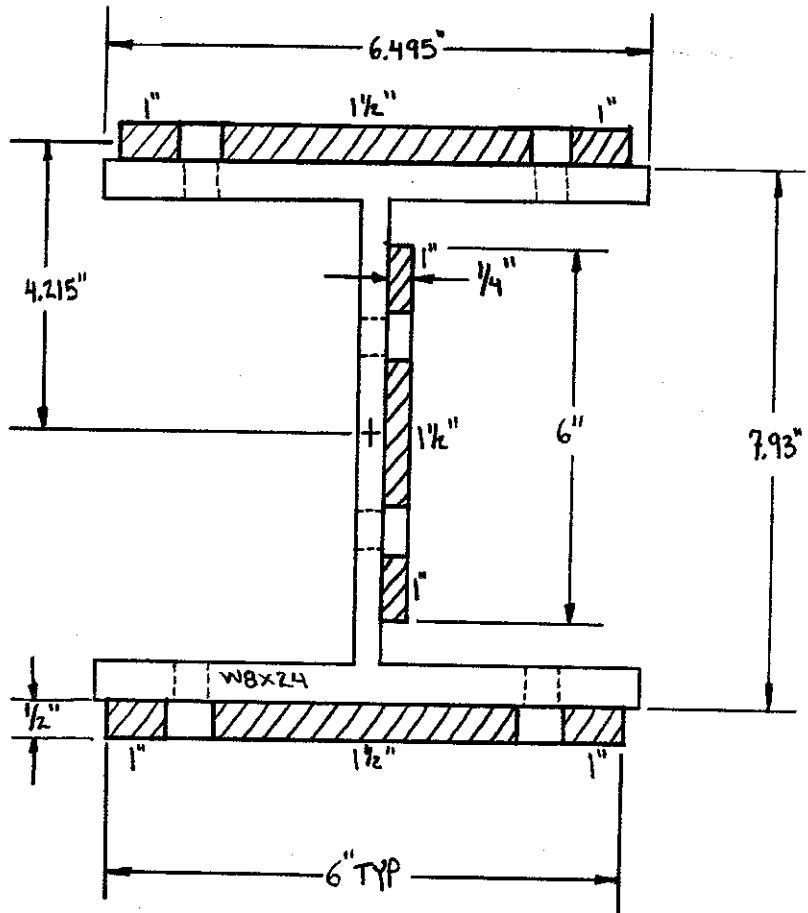
**STRUCTURAL
AND
MECHANICAL
DESIGN**

**W8X24
CONNECTION
JOINTS**

W8x24 Connection Joints

A system of three plates, two 1/2" thick plates and one 1/4" thick plate were used to connect adjoining frames of the inner, conical frame. These bolted on plates must be able to resist the forces and moments exerted on them. The weakest portion of each plate is through a plane passing through the bolt hole locations of each plate.

The moment of inertia and the section modulus were calculated for the cross section of the three plates. Maximum combined stress were calculated for all joints where "Frame Analysis" had indicated high stress. The maximum combined stress was 17,714 psi which is lower than the allowable 21,600 psi.



$$I_z = 65.4923 \text{ in}^4$$

$$I_y = 13.008 \text{ in}^4$$

$$C_z = 4.465 \text{ in}$$

$$C_y = 3 \text{ in}$$

$$S = \frac{I}{C}$$

$$S_z = 14.667 \text{ in}^3$$

$$S_y = 4.336 \text{ in}^3$$

$$\text{Area}_{\text{Top/Bot plate}} = 1.75 \text{ in}^2$$

$$A_{\text{Total}} = 4.375 \text{ in}^2$$

FORCES & MOMENTS

Element #	F _x	F _y	F _z	Torsion	M _y	M _z
19	-14840	-1691	-1366	-4180	-11830	45370
20	-25130	-2738	-2332	-6601	-18240	70100
21	-14900	1230	1214	1419	-11550	45610
22	-25210	2198	2028	2042	-17790	70490
39	-13540	-2768	-1552	-7842	-10320	61450
40	-23550	-3626	-3250	-14270	-14190	96260
41	-13880	-78	824	901	-9637	62050
42	-24010	1030	862	-152	-12780	97510
59	-16580	-1974	-1463	-9273	-12070	90490
60 (Largest)	-24750	-1981	-2166	-12670	-14860	125100
61	-16650	1300	1428	4180	-11560	90930
62	-24670	2846	2097	6134	-14220	125700

Units:
lbs
in-lb

continued

CONT. COMPLEX UPGRADE
Conical FrameStress Calculations on
W2x24, Joints; Plate Connections
Don Mitchell x4710
9/23/91

P2/2

Element #	Torsion Force Couple (lbs)	Torsion Shear $T_y = T_z$	STRESSES				
			P/A (in²)	σ_y	σ_z	τ_y	τ_z
19	496	283	3392	2728	3093	387	31
20	783	447	5744	4207	4779	626	533
21	168	96	3406	2664	3110	281	277
22	242	138	5762	4103	4806	502	464
39	930	532	3095	2380	4190	633	355
40	1693	967	5383	3273	6563	829	743
41	107	61	3173	2223	4231	18	188
42	18	10	5488	2947	6648	235	197
59	1100	629	3790	2784	6169	451	334
60	1503	859	5657	3427	8529	453	495
61	496	283	3806	2666	6199	297	326
62	728	416	5639	3280	8570	651	479

COMBINED STRESS

Element #

stress

$$\sigma_{\text{Comb.}} = \sqrt{(\sigma_x + \sigma_y + \sigma_z)^2 + (T_y + T_z)^2 + (\tau_x + \tau_z)^2}$$

19 9243

20 14802

21 9195

22 14697

39 9775

40 15420

41 9631

42 15086

59 12825

60 17714

61 12699

62 17544



Max. Stress < .6Fy = 21,600 psi

∴ Joint Connection Plates
are adequate.

CDF CMEX UPGRADE Maximum Stress Loads Don Mitchell x 4710.
 W8x24 Joints + Bolts on A325 3/4" Bolts 12/27/91

9 Chamber Frame

Question: Can the bolts that hold together the W8x24 joints withstand the forces and moments if the joint is looked at as a bearing type connection rather than a friction type connection?

Solution: The worst case joint is located at NODE # 71.

The forces and moments at this joint are:

$$F_x = 24750 \text{ lbs}$$

$$M_x = 12670 \text{ in-lb}$$

$$F_y = 1981 \text{ lbs}$$

$$M_y = 14860 \text{ in-lb}$$

$$F_z = 2166 \text{ lbs}$$

$$M_z = 125100 \text{ in-lb}$$

The moments are broken down into "couple forces" which either apply tensile or shear forces.

These forces acting on a A325, 3/4" bolt create tensile or shear stresses.

The calculated stress per bolt is then compared to the allowable stress.

(Calculations are shown on the next two pages)

Conclusion:

The maximum tensile stress is: 22,992 psi }

The allowable tensile stress is: 35,291 psi } OK

The maximum shear stress is: 12,542 psi }

The allowable shear stress is: 20,400 psi } OK

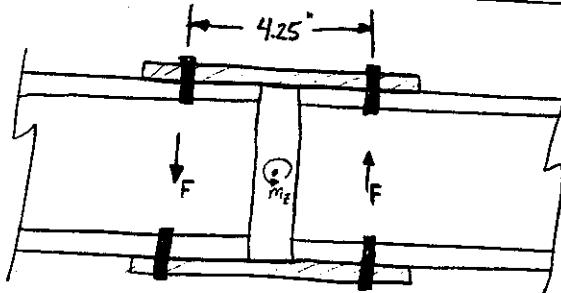
Tension Load = 10,158 lbs }

Allowable tension Load = 19,400 lbs } OK

Shear Load = 5,541 lbs }

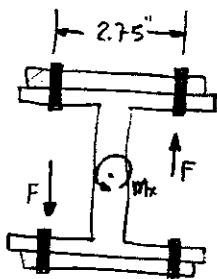
Allowable shear load = 9,300 lbs } OK

∴ A325 3/4" UNC bolts are adequate for a bearing type connection.



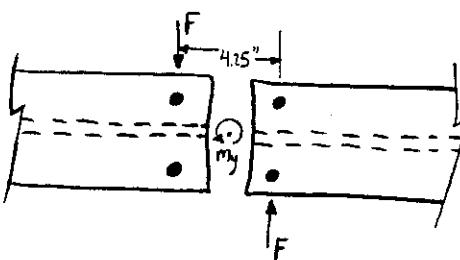
$$F = \frac{M_x}{D} = \frac{125100 \text{ in-lb}}{4.25 \text{ in}} = 29,435 \text{ #}$$

Divided by four bolts yields 7359 lbs/bolt (tension/compression)



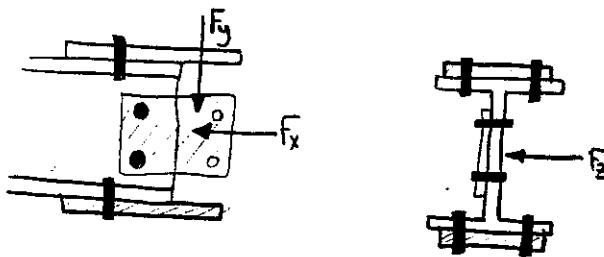
$$F = \frac{M_x}{D} = \frac{12670}{2.75} = 4,607 \text{ #}$$

Divided by 2 bolts yields 2304 lbs/bolt (tension/compression)



$$F = \frac{M_y}{D} = \frac{14860}{4.15} = 3496 \text{ #}$$

Divided by four bolts yields 874 lbs/bolt (shear)



$$\frac{F_y}{4} = \frac{1981}{4} = \underline{\underline{495 \text{ #}}} \text{ (tens./Comp)}$$

$$\frac{F_x}{6} = \frac{24750}{6} = \underline{\underline{4125 \text{ #}}} \text{ (Shear)}$$

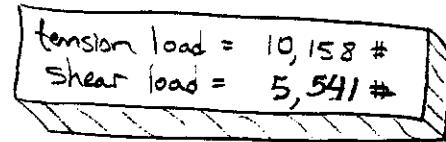
$$\frac{F_x}{4} = \frac{2166}{4} = \underline{\underline{542 \text{ #}}} \text{ (Shear)}$$

Total Shear and Tension Loads: (per bolt)Tension

$$7359 + 2304 + 495 = 10,158 \text{ #}$$

Shear

$$874 + 4125 + 542 = 5,541 \text{ #}$$



From the "Manual of Steel Construction" 9th ed., the allowable loads for shear and tension on a A325, 3/4" Bolt, Standard Bearing Connection, are:

$$\text{tensile load (max)} = 19,400 \text{ #}$$

$$\text{shear load (max)} = 9,300 \text{ #}$$

The applied loads are less than the allowable loads. \therefore Bolts are adequate under this loading check. But, are maximum bolt stresses below the allowable?

Maximum Bolt stress:

$$\text{Tensile Stress: } \frac{F_t}{\text{Bolt Area}} = f_{t\max} = \frac{10,158 \text{ #}}{.4418 \text{ in}^2} = 22,992.3 \text{ psi}$$

$$\text{Shear Stress: } \frac{F_v}{\text{Area}} = f_{v\max} = \frac{5541 \text{ #}}{.4418 \text{ in}^2} = 12,541.9 \text{ psi}$$

Allowable Stress:

$$\begin{aligned} \text{Allowable tensile stress} &= \sqrt{(44)^2 - 4.39 f_v^2} \\ &= \sqrt{(44)^2 - 4.39 (12.54)^2} \\ &= 35.291 \text{ KSI} = \underline{\underline{35,291 \text{ psi}}} \end{aligned}$$

$$\text{Allowable shear stress} = .17 F_u = (.17)(120,000 \text{ psi}) = \underline{\underline{20,400 \text{ psi}}}$$

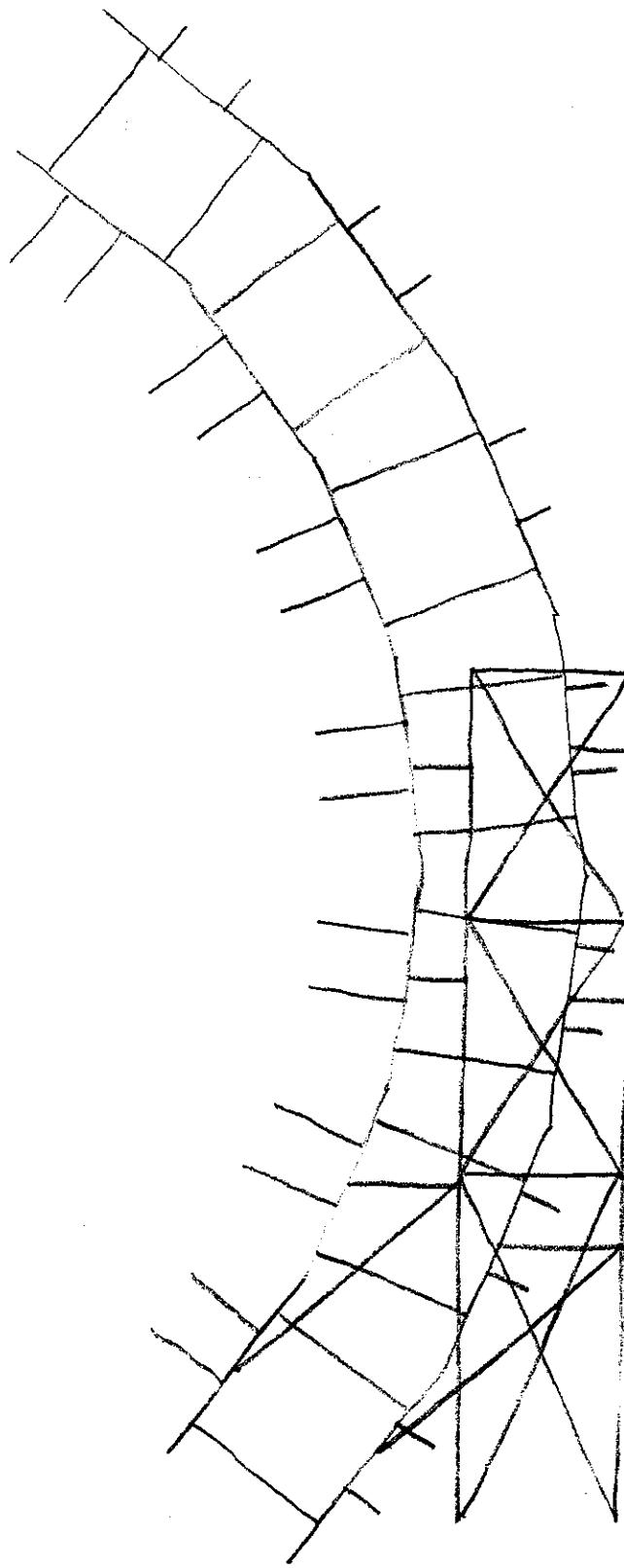
The actual stresses are below the allowables, therefore the bolts are adequate.

**CDF
CMEX
UPGRADE**

**STRUCTURAL
AND
MECHANICAL
DESIGN**

**7 Chamber Stress Analysis,
Bolt Analysis,
and Connection Plate Analysis
without Super-Structure Support Frame**

7 Chamber Frame



A stress analysis was performed on the CMEX conical frame with 9 chambers and was presented to the safety panel. These 9 chambers were supported by an outer supporting frame which connects to each section of the inner conical frame.

In this 7 chamber frame analysis, only the first four inner frame sections will be supported by the outer support frame. This means that the super structure needed to support 9 chambers will not be used to support 7 chambers. In other words, in this 7 chamber analysis, the top 3 chambers will be connected to each other ONLY and not to an outer supporting frame.

Question #1: Are the stresses in the inner conical frame below the allowable stress?

Question #2: Can the bolted joint connections handle the load if the slip-critical connection was actually a bearing type connection?

Question #3: Are the plates in the bolted joint connections below the allowable stress?

In the 9 chamber configuration, the answers to these questions was "Yes." However, in the 7 chamber scheme since the super structure will not be used and 3 chambers will be partially un-supported, the answers to these questions is not intuitive.

The analysis will begin with a conservative look at the stresses in the 7 chamber inner conical frame.

Wide-flange Stress Calculations: (Question #1 solution)

Instead of calculating stresses for each wide-flange, the maximum forces and moments out of all the wide-flange members will be used and assumed to be acting all on one beam. If the combined stress is below the allowable stress of 21,600 psi, then it can be assumed that all wide-flange elements are below the allowable stress. If not, an individual analysis of each beam will be performed.

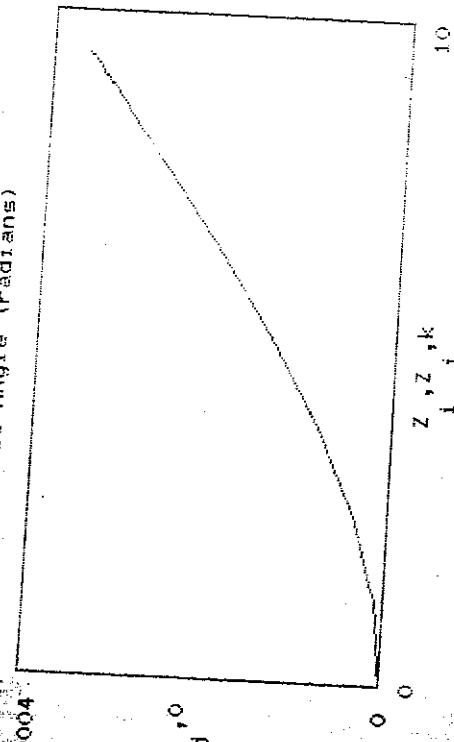
θ_m

7 Chamber Frame Torsional Analysis on W8x24 Beam

TWIST ANGLE AND ITS DERIVATIVES

Twist Angle (radians)

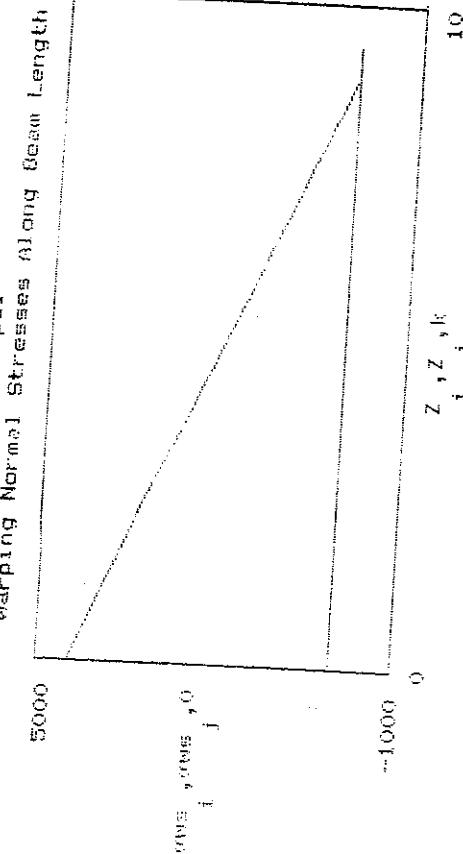
0.004



TORSIONAL ANALYSIS OF WIDE-FLANGE MEMBER
 $\alpha = 0.95$

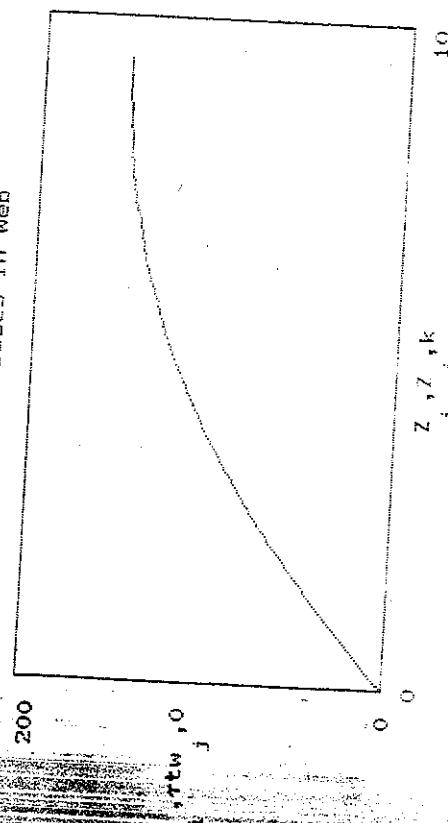
TORSIONAL STRESSES

Warping Normal Stresses

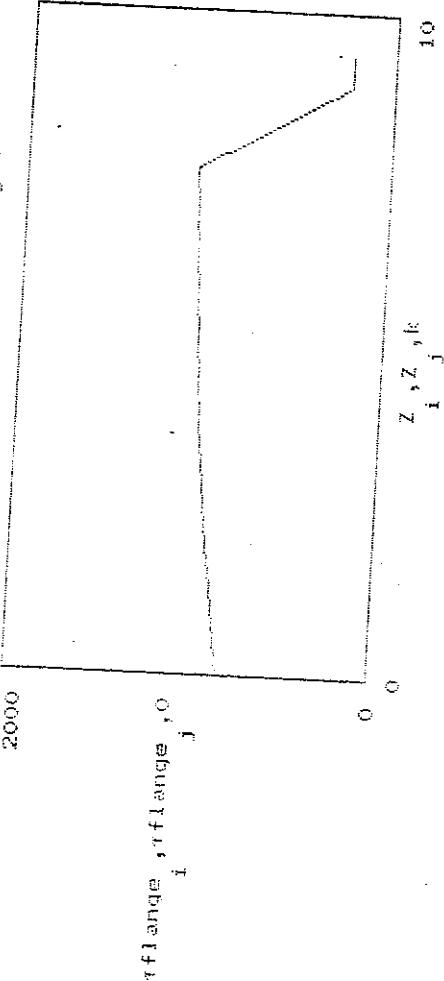


Torsional Shear Stresses

Shear Stresses in Web



Shear Stress in Flanges



$$f_a = 1.551 \cdot 10^{-3} \text{ psi} \quad (\text{axial stress})$$

Situations Parameters

$$E = 2.9 \cdot 10^7$$

Modulus of Elasticity

New Shear Stress

Flange Shear Stress

Twist Angle at Flange

Radii of Curvature

Twist Angle at Web

Flange Shear Stress

Twist Angle at Web

Flange Shear Stress

Twist Angle at Flange

Radii of Curvature

Twist Angle at Web

Flange Shear Stress

Twist Angle at Flange

Radii of Curvature

Twist Angle at Web

Flange Shear Stress

Twist Angle at Flange

Radii of Curvature

Twist Angle at Web

Flange Shear Stress

Twist Angle at Flange

Radii of Curvature

Twist Angle at Web

Flange Shear Stress

Twist Angle at Flange

Radii of Curvature

Twist Angle at Web

Flange Shear Stress

Twist Angle at Flange

Radii of Curvature

Twist Angle at Web

Flange Shear Stress

Twist Angle at Flange

Radii of Curvature

Twist Angle at Web

Flange Shear Stress

Twist Angle at Flange

Radii of Curvature

Twist Angle at Web

Flange Shear Stress

Twist Angle at Flange

Radii of Curvature

Twist Angle at Web

Position

Z Position

m Position

0 Position

-6 Position

1.272 Position

2.543 Position

3.815 Position

5.087 Position

6.358 Position

7.63 Position

8.902 Position

8.96 Position

9.077 Position

9.136 Position

9.194 Position

9.253 Position

9.311 Position

9.37 Position

9.43 Position

9.49 Position

9.55 Position

9.61 Position

9.67 Position

9.73 Position

9.79 Position

9.85 Position

9.91 Position

9.97 Position

1.034 Position

1.048 Position

1.062 Position

1.076 Position

1.090 Position

1.104 Position

1.118 Position

1.132 Position

1.146 Position

1.160 Position

1.174 Position

1.188 Position

1.202 Position

1.216 Position

1.230 Position

1.244 Position

Situations Parameters

Modulus of Rigidity

$G = 1.12 \cdot 10^7$

Modulus of Rigidity

$P = 1.098 \cdot 10^4$

Axial Load (lb)

$M = 1.082 \cdot 10^4$

Torsional Moment (in-lb)

$J = 0.35$

Torsional Constant

$W_{HO} = 1.22 \cdot 10^4$

Normalized Warping Constant

$S_W = 7.94$

Warping Static Moment

$A = 7.08$

Area of Cross-Section

Wide-Flange Dimensions (inches)

$L_u = 9.37$

Unsupported Length of Beam

$t_w = 0.245$

Web Thickness

Flange Thickness

$h = 7.93$

Overall Height of Wide-Flange

$t_f = 0.4$

Flange Thickness

$h_f = 6.495$

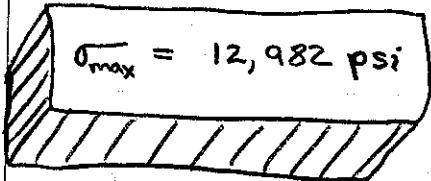
Overall Width of Wide-Flange

F_x max	F_y max	F_z max	M_x max	M_y max	M_z max
10980 lbs	1487 lbs	1323 lbs	10820 in-lb torsion	10000 in-lb	106800 in-lb
σ_x	γ_y	γ_z	σ_{ws}	γ_f	σ_y
F_x/A	F_y/A	F_z/A			M_y/S_y
$10980 / 7.08 \text{ in}^2$	$1487 / 7.08 \text{ in}^2$	$1323 / 7.08 \text{ in}^2$			$10000 / 5.63 \text{ in}^2$
1551 psi	210 psi	187 psi	4425 psi	1048 psi	1776 psi
					5110 psi

Combined stress:

$$\sigma_{\max} = \sqrt{(\sigma_x + \sigma_y + \sigma_z + \sigma_{ws})^2 + (\gamma_y + \gamma_f)^2 + (\gamma_z + \gamma_f)^2}$$

$$\sigma_{\max} = \sqrt{(1551 + 1776 + 5110 + 4425)^2 + (210 + 1048)^2 + (187 + 1048)^2}$$



$$\sigma_{\max} = 12,982 \text{ psi}$$

OK

$$F_a = .6 F_y = .6 (36,000 \text{ psi}) \\ = 21,600 \text{ psi}$$

$$\sigma_{\max} < F_a$$

Therefore, it can be assumed that all wide-flange members are strong enough even in a #5, #6, #7 unsupported chamber configuration.

Bearing type bolted connection: (Question #2 solution)

Similar to the analysis of combined stress in the W8x24 members, this analysis of bolt stress will use the maximum forces and maximum moments and assume that these forces and moments all act on one joint. If the bolts are strong enough to withstand these forces, then it can be assumed that all the bolted connections are adequate.

The maximum forces and moments are:

$$F_x = 10980 \text{ lbs}$$

$$M_x = 10,820 \text{ in-lb (torsion)}$$

$$F_y = 1487 \text{ lbs}$$

$$M_y = 10,000 \text{ in-lb (bending)}$$

$$F_z = 1323 \text{ lbs}$$

$$M_z = 106,800 \text{ in-lb (bending)}$$

The moments are broken down into "couple forces" which either apply tensile, compressive, or shear forces.

These forces acting on a A325, 3/4" bolt create tensile, compressive, or shear stresses.

The calculated stress per bolt is then compared to the allowable stress.

(Calculations are shown on the next two pages)

Conclusion: (Bolted Connections)

The maximum tensile stress is: 19,513 psi

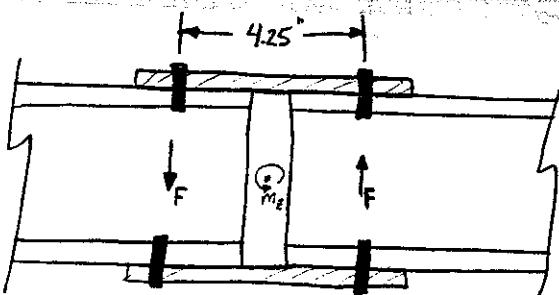
The allowable tensile stress is: 42,020 psi

The maximum shear stress is: 6,222 psi

The allowable shear stress is: 20,400 psi

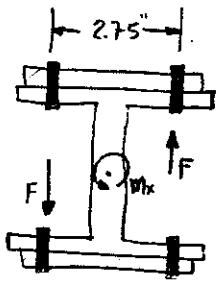
$$\left. \begin{array}{l} \text{Tension Load} = 8,621 \text{ lbs} \\ \text{Allowable tensile load} = 19,400 \text{ lbs} \end{array} \right\} \text{OK} \quad \left. \begin{array}{l} \text{Shear Load} = 2,749 \text{ lbs} \\ \text{allowable shear load} = 9,300 \text{ lbs} \end{array} \right\} \text{OK}$$

\therefore A325, 3/4" Bolts are adequate for a bearing type connection even if the friction connection does not hold.



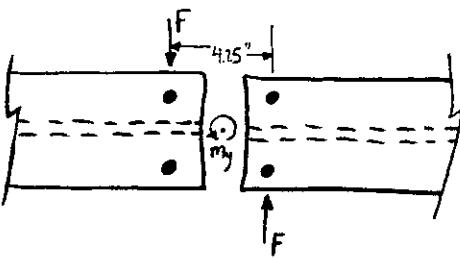
$$F = \frac{M_x}{D} = \frac{106,800 \text{ in-lb}}{4.25 \text{ in}} = 25,129 \text{ lbs}$$

Divided by 4 bolts yields 6,282 lbs/bolt in tension/compression



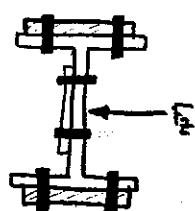
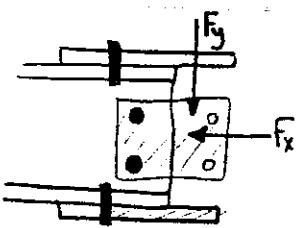
$$F = \frac{M_x}{D} = \frac{10,820 \text{ in-lb}}{2.75 \text{ in}} = 3935 \text{ lbs}$$

Divided by 2 bolts yields 1,967 lbs/bolt in tension/compression



$$F = \frac{M_y}{D} = \frac{10,000 \text{ in-lb}}{4.25 \text{ in}} = 2353 \text{ lbs}$$

Divided by 4 bolts yields 588 lbs/bolt in shear



$$\frac{F_y}{4} = \frac{1487 \text{ lbs}}{4} = 372 \text{ lbs (tens/comp)}$$

$$\frac{F_x}{6} = \frac{10980 \text{ lbs}}{6} = 1830 \text{ lbs (shear)}$$

$$\frac{F_x}{4} = \frac{1323 \text{ lbs}}{4} = 331 \text{ lbs (shear)}$$

Total shear and tension loads per bolt:tension

$$6282 + 1967 + 372 = \underline{8621 \text{ lbs}}$$

shear

$$588 + 1830 + 331 = \underline{2749 \text{ lbs}}$$

tension load per bolt = 8,621 lbs
shear load per bolt = 2,749 lbs

From the "Manual of Steel Construction", 9th ed., the allowable loads for shear and tension on a A325, 3/4" bolt, Standard Bearing Connection, are:

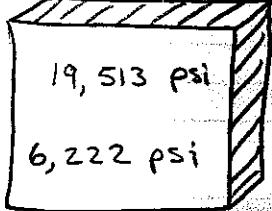
$$\text{tensile load (max)} = 19,400 \text{ lbs}$$

$$\text{shear load (max)} = 9,300 \text{ lbs}$$

The applied loads are less than the allowable loads. Therefore, Bolts are adequate under this loading scheme. But, are maximum bolt stresses below the allowable?

Maximum Bolt Stress:

$$(f_{t\max}) \text{Tensile stress: } \frac{F_t}{\text{Bolt Area}} = \frac{8,621 \text{ lbs}}{.4418 \text{ in}^2} =$$



$$19,513 \text{ psi}$$

$$(f_{v\max}) \text{Shear stress: } \frac{F_v}{\text{Bolt Area}} = \frac{2,749 \text{ lbs}}{.4418 \text{ in}^2} =$$

$$6,222 \text{ psi}$$

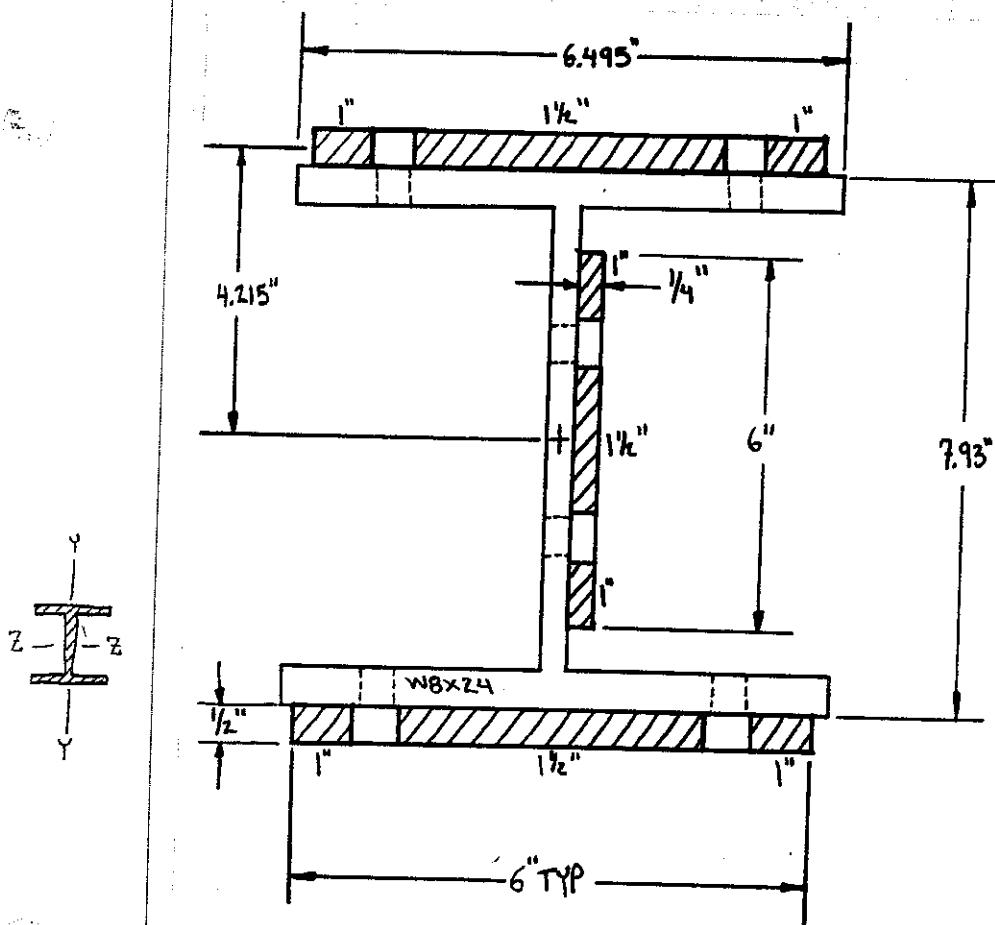
Allowable Bolt Stress:

$$\begin{aligned} \text{Allowable tensile stress} &= \sqrt{(44)^2 - 4.39 f_y^2} \\ &= \sqrt{(44)^2 - 4.39 (6.222)^2} \\ &= 42.02 \text{ KSI} = \underline{\underline{42,020 \text{ psi}}} \end{aligned}$$

$$\begin{aligned} \text{Allowable shear stress} &= .17 F_u = (.17)(120,000 \text{ psi}) \\ &= \underline{\underline{20,400 \text{ psi}}} \end{aligned}$$

The actual stresses are below the allowable stresses.

Therefore, for the 7 chamber (non super-structure support), the bolts are adequate even if the friction connections fail.



$$I_z = 65.4923 \text{ in}^4$$

$$I_y = 13.008 \text{ in}^4$$

$$C_z = 4.465"$$

$$C_y = 3"$$

$$S = \frac{I}{C}$$

$$S_z = 14.667 \text{ in}^3$$

$$S_y = 4.336 \text{ in}^3$$

$$\text{Area}_{\text{Top/Bot. plate}} = 1.75 \text{ in}^2$$

$$A_{\text{Total}} = 4.375 \text{ in}^2$$

Plate Connection Stress Calculations: (Question #3 Solution)

Maximum forces and moments from all joints will be combined and assumed to act all on one joint. If the connection plates can withstand this load application then it will be assumed that all connection joints are strong enough.

Maximum Forces and Moments:

$$F_x = 10980 \text{ lbs}$$

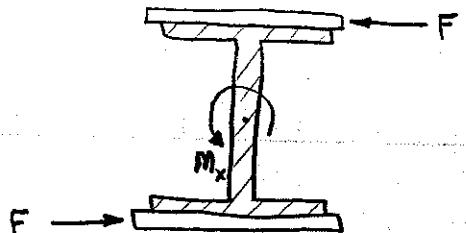
$$M_x = 10820 \text{ in-lb (torsion)}$$

$$F_y = 1487 \text{ lbs}$$

$$M_y = 10000 \text{ in-lb}$$

$$F_z = 1323 \text{ lbs}$$

$$M_z = 106,800 \text{ in-lb}$$



$$F = \frac{M_x}{D} = \frac{10820}{7.93} = 1364 \text{ lbs}$$

Torsion Shear: $\gamma_y' = \gamma_z'$

$$\gamma_y' = \gamma_z' = \frac{F}{A_{\text{Top plate}}} = \frac{1364 \text{ lbs}}{1.75 \text{ in}^2} = 780 \text{ psi}$$

Axial Stress: σ_x

$$\sigma_x = \frac{F_x}{A} = \frac{10980 \text{ lbs}}{4.375 \text{ in}^2} = 2,510 \text{ psi}$$

Bending Stress: σ_y , σ_z

$$\sigma_y = \frac{M_y}{S_y} = \frac{10000 \text{ in-lb}}{4.336 \text{ in}^3} = 2,306 \text{ psi}$$

$$\sigma_z = \frac{M_z}{S_z} = \frac{106,800 \text{ in-lb}}{14.667 \text{ in}^3} = 7,282 \text{ psi}$$

Shear Stress: τ_y , τ_z

$$\tau_y = \frac{F_y}{A} = \frac{1487 \text{ lbs}}{4.375 \text{ in}^2} = 340 \text{ psi}$$

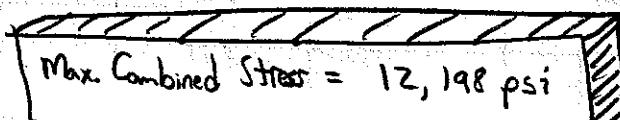
$$\tau_z = \frac{F_z}{A} = \frac{1323 \text{ lbs}}{4.375 \text{ in}^2} = 302 \text{ psi}$$

Combined Stress:

$$\sigma_{\text{comb.}} = \sqrt{(\sigma_x + \sigma_y + \sigma_z)^2 + (\tau_y + \tau_z)^2 + (\tau_z + \tau_z')^2}$$

$$= \sqrt{(2510 + 2306 + 7282)^2 + (340 + 780)^2 + (302 + 780)^2}$$

$$= 12,198 \text{ psi}$$



$$\text{Max. Allowable Stress} = F_a = .6 F_y = (.6)(36,000 \text{ psi}) = 21,600 \text{ psi}$$

$$\sigma_{\text{max}} < F_a$$

\therefore all connection plates are adequate.